2-way proportional throttle valve for block installation

Types FES; FESE

Sizes 25 to 63
Component series 3X
Maximum operating pressure 315 bar
Maximum flow 1800 l/min at $\Delta p = 10$ bar

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<td>16</td>
</tr>
</tbody>
</table>

Features

- Pilot operated 2-way proportional throttle valve for block installation
- Installation dimensions to DIN ISO 7368
- Orifice spool electrically closed-loop position controlled
- Flow in both directions
- In the event of a power failure, cable break or withdrawal of the enable, the orifice spool automatically moves to the seated position and blocks the flow in both directions
- Can be used in conjunction with a pressure compensator for pressure-compensated flow control
- Type FES for external control electronics (separate order), see page 5
- Type FESE: completely matched unit with integrated electronics (OBE), optionally available with voltage or current interface

Information on available spare parts:
www.boschrexroth.com/spc
### Ordering code

<table>
<thead>
<tr>
<th>FES</th>
<th>C</th>
<th>A</th>
<th>3X/</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

**For external control electronics = No code**

With integrated electronics (OBE) = E

- Size 25 = 25
- Size 32 = 32
- Size 40 = 40
- Size 50 = 50
- Size 63 = 63

Kit = C

**Direction of flow**

- A to B (X connected to A) = A
- B to A (X connected to B) = B

Component series 30 to 39 = 3X

(30 to 39: unchanged installation and connection dimensions)

**Flow characteristics "linear"**

1. **Size 25** up to 315 l/min = 315L
2. **Size 32** up to 450 l/min = 450L
3. **Size 40** up to 670 l/min = 670L
4. **Size 50** up to 1400 l/min = 1400L
5. **Size 63** up to 1800 l/min = 1800L

1 Nominal flow in L/min at \( \Delta p \) 10 bar between ports A and B (see also hydraulic technical data on page 4)

### Standard types

<table>
<thead>
<tr>
<th>Type</th>
<th>Material no.</th>
</tr>
</thead>
<tbody>
<tr>
<td>FESE 25 CA-3X/315LK0B1M</td>
<td>R900973604</td>
</tr>
<tr>
<td>FESE 32 CA-3X/450LK0B1M</td>
<td>R900973605</td>
</tr>
<tr>
<td>FESE 40 CA-3X/670LK0B1M</td>
<td>R900973607</td>
</tr>
<tr>
<td>FESE 50 CA-3X/1400LK0B1M</td>
<td>R900954504</td>
</tr>
<tr>
<td>FESE 63 CA-3X/1800LK0B1M</td>
<td>R900954505</td>
</tr>
</tbody>
</table>

### Symbols

**Simplified**

- FES .. CA-3X/...
- FESE .. CA-3X/...

**Direction of flow:**

- A to B (X connected with A)
- B to A (X connected with B)

**Detailed**

(Example of FES)

- A = service port
- B = service port
- X = pilot oil supply
- Y = pilot oil drain

---

Further details in clear text

### Seal material

- M = NBR seals, suitable for mineral oil (HL, HLP) to DIN 51524
- V = FKM seals

### Electronics interface

(see page 7)

- B1 = Command value input 0 to 10 V/actual value output 0 to –10 V
- G1 = Command value input 4 to 20 mA/actual value output 4 to 20 mA

**No code** = For FES for external control electronics

### Electrical connection

**For FES:**

- K4 = Without cable sockets, with component plug to DIN EN 175301-803 for proportional solenoid and GSA20 made by Hirschmann for position transducer
- Cable sockets – separate order, see page 6

**For FESE:**

- K0 = Without cable socket, with component plug to DIN 43651, cable socket – separate order, see page 7
Function, section

Valve types FES(E) are pilot operated 2-way proportional throttle valves for block installation for the infinitely variable control of a flow.

Technical structure:
The valve consists of four main assemblies:
- Cover (1) with mounting face for pilot oil ports.
- Main valve (2) with orifice spool (3).
- Pilot valve (4) with proportional solenoid (5).
- Integrated control electronics (6) (not provided for type FES) with position transducer (7).

General function:
- Command value-related closed-loop position control of orifice spool (3) and therefore defined opening of orifice (8).
- The flow depends on the $\Delta p$ across orifice (8) and the position of orifice spool (3).
- Actual value acquisition of the position of orifice spool (3) by position transducer (7); command/actual value comparison in electronics (6); deviations are conditioned and passed on to proportional solenoid (5) of pilot valve (4) in the form of a control output for correcting the position of orifice spool (3).
- Area ratio of area (14) to area (15) = 2 : 1 for size 25; 32; 40, and 1.6 : 1 for size 50; 63.
- Direction of flow $A \rightarrow B$ (connect X with A);
direction of flow $B \rightarrow A$ (connect X with B);
external pilot oil supply via X possible.
- When the enable is withdrawn, orifice spool (3) moves onto valve seat (9) and closes the direction of flow $A \leftrightarrow B$ leak-free. Spool seal (11) ensures the leak-free isolation of port B from control chamber (12); with internal pilot oil supply, take leakage oil from X via the pilot valve to Y into account!
- Orifice spool position is already controlled at a command value of 0 V or 4 mA, with orifice (8) still being in the positive overlap position.

Function of opening orifice spool:
(Assumption: flow $A \rightarrow B$ and A connected with X)
- Proportional solenoid (5) shifts pilot spool (4) against spring (3) and opens the connection between control chamber (12) and Y; the pressure in control chamber (12) is reduced and orifice spool (3) moved to the direction of opening by the pressure in A that acts on area (5) plus the pressure in B that acts on the annulus area (6).

Function of closing orifice spool:
(Assumption: flow $A \rightarrow B$ and A connected with X)
- Current reduced in proportional solenoid (5); spring (3) shifts pilot spool (4) against the proportional solenoid and opens the connection between X and control chamber (12); the pressure acting on area (14) plus spring force (10) shift orifice spool (3) in the closing direction.

Flow control function:
- In conjunction with a pressure compensator, can be used for the pressure-compensated control of a flow.

Failure of supply voltage:
- The integrated electronics de-energises the solenoid in the event of a supply voltage failure or cable break in position transducer (7).
- The spool is shifted to valve seat (9) by the pressure applied to pilot port X plus spring force (10) and blocks the flow $A \rightarrow B$.

⚠️ Caution: A voltage supply failure results in a sudden standstill of the controlled axis. Accelerations that can occur in conjunction with this can cause damage to machines!
## Technical data (for applications outside these parameters, please consult us!)

### General

<table>
<thead>
<tr>
<th>Size</th>
<th>25</th>
<th>32</th>
<th>40</th>
<th>50</th>
<th>63</th>
</tr>
</thead>
<tbody>
<tr>
<td><strong>Weight</strong></td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– FES kg</td>
<td>3.8</td>
<td>5.5</td>
<td>8.2</td>
<td>12.5</td>
<td>21</td>
</tr>
<tr>
<td>– FESE kg</td>
<td>4</td>
<td>5.7</td>
<td>8.4</td>
<td>12.7</td>
<td>21.2</td>
</tr>
<tr>
<td><strong>Installation orientation</strong></td>
<td>Optional</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Storage temperature range</strong></td>
<td>ºC</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– FES</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– FESE</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Ambient temperature range</strong></td>
<td>ºC</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– FES</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– FESE</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

### Hydraulic (measured with HLP 46; \( \vartheta_{\text{oil}} = 40 ^\circ \text{C} \pm 5 ^\circ \text{C} \))

<table>
<thead>
<tr>
<th>Size</th>
<th>25</th>
<th>32</th>
<th>40</th>
<th>50</th>
<th>63</th>
</tr>
</thead>
<tbody>
<tr>
<td><strong>Max. operating pressure – Ports A, B</strong> bar</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Max. pilot pressure – Port X</strong> bar</td>
<td>315</td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Min. inlet pressure – in A (direction of flow A → B)</strong> bar</td>
<td>12</td>
<td>15</td>
<td>15</td>
<td>20</td>
<td>20</td>
</tr>
<tr>
<td><strong>Max. flow ( q_{\text{Vmax}} ) of main valve at ( \Delta p ) 10 bar</strong> l/min</td>
<td>360</td>
<td>480</td>
<td>680</td>
<td>1400</td>
<td>1800</td>
</tr>
<tr>
<td><strong>Pilot oil volume for switching process from seated position → 100%</strong> cm³</td>
<td>3.9</td>
<td>7.6</td>
<td>12</td>
<td>23.4</td>
<td>52</td>
</tr>
<tr>
<td><strong>Max. pilot oil volume in port Y:</strong></td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– With stepped input signal l/min</td>
<td>5.0</td>
<td>6.5</td>
<td>10</td>
<td>12</td>
<td>17</td>
</tr>
<tr>
<td><strong>Direction of flow</strong></td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– Internal pilot oil supply A → B Connect A to X</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>B → A Connect B to X</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– External pilot oil supply A → B Pressure at X &gt; pressure in A</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>B → A Pressure at X &gt; pressure in B</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Leakage fluid</strong></td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– State: Command value 0 V or 4 mA, from A → B / B → A in dependence on ( \Delta p )</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>from A → X / B → X via pilot control to Y at ( p = 315 ) bar See characteristic curves on pages 9 to 14</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>– State: Enable inactive ( \vartheta_{\text{oil}} &lt; 0.2 ) l/min at ( \Delta p = 315 ) bar</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>( q_{\vartheta} )</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>A → B / B → A leak-free isolation</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td><strong>Caution!</strong></td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>In the case of internal pilot oil supply, observe leakage from A or B to X via the pilot valve to Y. ( q_{\vartheta} &lt; 0.2 ) l/min at ( \Delta p = 315 ) bar</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>With external pilot oil supply to X, this fluid loss caused by leakage from A or B can be avoided. The external pressure at X must be ( \geq ) the pressure in A with direction of flow A → B and ( \geq ) the pressure in B with direction of flow B → A.</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

### Hydraulic fluid

Mineral oil (HL, HLP) to DIN 51524; further hydraulic fluids on enquiry!

| **Hydraulic fluid temperature range** | ºC | | | | |
| **Viscosity range** | mm²/s | | | | |
| **Max. permissible degree of contamination of the hydr. fluid** | | | | | |
| Cleanliness class – Pilot valve | Class 17/15/12 | | | | |
| to ISO 4406 (c) – Main valve | Class 20/18/15/ | | | | |
| **Hysteresis** | % | | | | |
| **Response sensitivity** | % | | | | |
| **Range of inversion** | % | | | | |

Hydraulic fluid temperature range – 20 to + 80

Viscosity range – 15 to 380

Max. permissible degree of contamination of the hydr. fluid

Cleanliness class – Pilot valve

Cleanliness class – Main valve

Hysteresis $< 0.2$

Response sensitivity $< 0.1$

Range of inversion $< 0.15$
Technical data (for applications outside these parameters, please consult us!)

Type FES – external control electronics

**Electrical, solenoid (pilot valve)**

<table>
<thead>
<tr>
<th>Type of voltage</th>
<th>V</th>
<th>24 DC</th>
</tr>
</thead>
<tbody>
<tr>
<td>Nominal current</td>
<td>mA</td>
<td>1000</td>
</tr>
<tr>
<td>Coil resistance</td>
<td>Ω</td>
<td>12.7</td>
</tr>
<tr>
<td>– Cold value at 20 °C</td>
<td></td>
<td></td>
</tr>
<tr>
<td>– Max. hot value</td>
<td></td>
<td>19.3</td>
</tr>
<tr>
<td>Duty cycle</td>
<td>%</td>
<td>100</td>
</tr>
</tbody>
</table>

**Electrical connection**

With component plug to DIN EN 7530-803

Cable socket to DIN EN 7530-803 2)

**Type of protection of the valve to EN 60529**

IP65 with cable socket mounted and locked

**Electrical, inductive position transducer (main stage; only for type FES)**

<table>
<thead>
<tr>
<th>Coil resistance</th>
<th>Total resistance of coils between 1 and 2</th>
<th>2 and ⊥</th>
<th>⊥ and 1</th>
</tr>
</thead>
<tbody>
<tr>
<td>at 20 °C (see Symbols on page 2)</td>
<td>Ω</td>
<td>31.5</td>
<td>45.5</td>
</tr>
<tr>
<td>Inductance</td>
<td>mH</td>
<td>6 to 8</td>
<td></td>
</tr>
<tr>
<td>Oscillator frequency</td>
<td>kHz</td>
<td>2.5</td>
<td></td>
</tr>
</tbody>
</table>

**Electrical connection**

With component plug GSA20 made by Hirschmann

Cable socket GM209N (Pg9) made by Hirschmann 3)

**Type of protection to EN 60529**

IP65 with cable socket mounted and locked

**Electrical position measuring system**

Differential throttle

**Control electronics (only for type FES; separate order)**

<table>
<thead>
<tr>
<th>Amplifier in Euro-card format</th>
<th>Size</th>
<th>25</th>
<th>32</th>
<th>40</th>
<th>50</th>
<th>63</th>
</tr>
</thead>
<tbody>
<tr>
<td>to data sheet RE 30117 analogue</td>
<td>VT-VRPA1-50</td>
<td>VT-VRPA1-51</td>
<td>VT-VRPA1-52</td>
<td></td>
<td></td>
<td></td>
</tr>
<tr>
<td>Amplifier of modular design to data sheet RE 29756 analogue</td>
<td>VT 11037</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

**Type FESE – integrated electronics (OBE)**

**Electrical**

| Current consumption | A | 1.3 |
| – Pulse load | A | 1.5 |
| Duty cycle | % | 100 |

**Electrical connection**

With component plug to DIN 43651

Cable socket to DIN 43651 11-pin + PE/Pg16 3)

**Type of protection of the valve**

IP65 with cable socket mounted and locked

**Control electronics**

Integrated in the valve (see page 8)

---

1) The cleanliness classes specified for components must be adhered to in hydraulic systems. Effective filtration prevents malfunction and, at the same time, prolongs the service life of components.

For the selection of filters, see data sheets RE 50070, RE 50076, RE 5008, RE 50086 and RE 50088.

2) Separate order, see page 6

3) Separate order, see page 7

---

**Note:** Details with regard to environment simulation testing in the fields of EMC (electromagnetic compatibility), climate and mechanical stress, see RE 29209-U (declaration on environmental compatibility).
Electrical connection, cable sockets (nominal dimensions in mm)

**Type FES** – for external control electronics

Connection to component plug

Connection to cable socket

Cable socket to DIN EN 175301-803
Separate order stating material no. **R901017011**
(plastic version)

Inductive position transducer

Cable socket GM209N (Pg9) made by Hirschmann
Separate order stating material no. **R900013674**
(plastic version)

Fixing screw M3
Tightening torque $M_T = 0.5 \text{ Nm}$
### Electrical connection, cable sockets (nominal dimensions in mm)

**Type FESE** – with integrated electronics (OBE)

Cable socket to DIN 43651/11-pin + PE/Pg16

Separate order stating material no. R900884671 (plastic version)

Assembly consisting of items 1 and 1.1 or items 1 and 1.2, type of protection IP65

---

#### Table: Pin Function and Conditions

<table>
<thead>
<tr>
<th>Pin</th>
<th>Function</th>
<th>Conditions</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>Operating voltage +UL</td>
<td>$U_O = 24$ VDC; $u_O(t)<em>{\text{max}} = 36$ V; $u_O(t)</em>{\text{min}} = 21.6$ V</td>
</tr>
<tr>
<td>2</td>
<td>Ground L0</td>
<td></td>
</tr>
<tr>
<td>3</td>
<td>Enable input / reference for pin 2</td>
<td>$\log 1 = 10$ V to $36$ V; $\log 0 = U &lt; 8$ V</td>
</tr>
<tr>
<td>4</td>
<td>Command value input</td>
<td>Voltage interface</td>
</tr>
<tr>
<td>5</td>
<td>Command value input, reference</td>
<td>Current interface</td>
</tr>
<tr>
<td>6</td>
<td>Actual value output</td>
<td>$0$ V to $10$ V ($I_{\text{max}} = 5$ mA)</td>
</tr>
<tr>
<td>7</td>
<td>Actual value output, reference</td>
<td></td>
</tr>
<tr>
<td>8</td>
<td>free</td>
<td></td>
</tr>
<tr>
<td>9</td>
<td>free</td>
<td></td>
</tr>
<tr>
<td>10</td>
<td>free</td>
<td></td>
</tr>
<tr>
<td>11</td>
<td>Ready for operation (output)</td>
<td>Valve not ready for operation: $U_{\text{pin11}} &lt; 8$ V; $U_{\text{pin11}} = U_O - 3$ V</td>
</tr>
<tr>
<td></td>
<td></td>
<td>Reference – pin 2: $I_{\text{max}}$ against $0$ V; $50$ mA;</td>
</tr>
<tr>
<td>PE</td>
<td>Protective conductor ⊥</td>
<td></td>
</tr>
</tbody>
</table>

---

**Note:**

- If you use one cable, combine item 1 with item 1.1
- If you use two cables, combine item 1 with item 1.2

---

**Recommended connecting cable**

- Up to 25 m $\rightarrow$ min. 0.75 mm² per wire
- Up to 50 m $\rightarrow$ min. 1.5 mm² per wire
- Connect shield to PE only on the supply side
### Integrated electronics (OBE) of type FESE

**Function**

1. **Making operation/disturbance characteristic:**
   After the supply voltage of 24 V was applied, the electronics is ready for operation, if the following conditions are fulfilled:
   - Operating voltage \( U_O > 18 \) VDC
   - The internal ± 7.5 V supply voltage is symmetrical
   - The connection to the position transducer is not interrupted.
   - The command value cable is not interrupted (only with 4 mA to 20 mA interface)
   If one of these conditions is not fulfilled, the controller and the output stage are blocked and the signal “ready for operation” is set to \(< 8\) V.

2. **Normal operation**
   When the enable is inactive (\(< 8\) V) and an optional command value is fed forward (0 to 10V or 4 to 20 mA) the orifice spool is in the seated position and blocks the flow from A to B.

By applying a voltage > 10 V to the enable, the position controller for the orifice spool and the output stage for the pilot valve are switched on. At the same time, the position controller (PID) compares the actual value of the orifice spool position with the applied command value, and a control output is fed to the output stage, which changes the solenoid current until the orifice spool position corresponds to the command value.

The actual value of the orifice spool position is sensed by an inductive position transducer. The signal of the latter is rectified by the demodulator and fed back to the PID-controller.

The following output signals are available on the plug:
   - Actual position value FESE.../...B1 (pin 6)
     - 0 V to – 10 V corresponds to 0 % to 100 % valve opening
     - Orifice spool in seated position \(\rightarrow\) actual value > 0.8 V
   - Actual position value FESE.../...G1 (pin 6)
     - 4 mA to 20 mA corresponds to 0 % to 100 % valve opening
     - Orifice spool in seated position \(\rightarrow\) actual value < 2.7 mA
   - Signal “ready for operation” (pin 11)
     - All conditions listed above are fulfilled \(\rightarrow\) > 10 V
     - One of the conditions is not fulfilled \(\rightarrow\) < 8 V

### Block circuit diagram / pin assignment of integrated electronics

1. **Command value**
2. **Actual value**
3. **Enable**
   - \(> 10\) V to 36 V = log 1
   - \(< 8\) V = log 0
4. **Ready for operation (output)**
   = \(U_O – 3\) V
   Not ready \(U < 8\) V
5. **Voltage supply**
   \(U_O(t)_{max} = 36\) V
   \(U_O(t)_{min} = 21.6\) V

1) With current version (4 mA to 20 mA), please observe:
   - Between connections 5 and 4, load = 100 Ω
   - Between connections 6 and 7, load \(\leq 500\) Ω

---

1. Input
2. Output
3. Fixed ramp
4. Position controller
5. Clock pulse
6. Current regulator
7. I/U converter
8. Output stage
9. Proportional solenoid
10. Position transducer
11. Oscillator / demodulator
12. Fault signal of position transducer
13. Power supply unit
14. Error signal in the case of \(+U_O\) undervoltage and asymmetry in the power supply unit
15. Cable break signal with current command value

---

Stroke \(s\) in mm

---
**Characteristic curves** (measured with HLP 46 and $\theta_{oil} = 40^\circ C \pm 5^\circ C$)

**Flow characteristic linear**

FES(E) 25 C…/315L… direction of flow A $\rightarrow$ B

![Graph showing flow characteristic linear for direction A to B.]

- 1 $\Delta p = 5$ bar
- 2 $\Delta p = 10$ bar
- 3 $\Delta p = 20$ bar
- 4 $\Delta p = 30$ bar
- 5 $\Delta p = 50$ bar
- 6 $\Delta p = 100$ bar

FES(E) 25 C…/315L… direction of flow B $\rightarrow$ A

![Graph showing flow characteristic linear for direction B to A.]

- 1 $\Delta p = 5$ bar
- 2 $\Delta p = 10$ bar
- 3 $\Delta p = 20$ bar
- 4 $\Delta p = 30$ bar
- 5 $\Delta p = 50$ bar
- 6 $\Delta p = 100$ bar

**Transient function in the case of stepped command value change**

Step responses
- 0 - 100 - 0 %
- 10 - 90 - 10 %
- 25 - 75 - 25 %

1) Measurement conditions
- Pressure in A = 50 bar
- Actuator in B closed ($p_A = p_B = 50$ bar)
- Pressure in A < 50 bar $\rightarrow$ actuating time is extended
- Pressure in A > 50 bar $\rightarrow$ actuating time is shortened
- The area ratio of the orifice spool has an influence on the actuating time as follows:
  - $\rightarrow$ Command value 0 $\rightarrow$ 100%: The actuating time becomes shorter, the higher the inlet pressure and the smaller the $\Delta p$ across the valve.
  - $\rightarrow$ Command value 100 $\rightarrow$ 0%: The actuating time becomes shorter, the higher the inlet pressure and the higher the $\Delta p$ across the valve.

**Leakage from A $\rightarrow$ B and B $\rightarrow$ A in dependence upon the pressure differential $\Delta p$ (command value 0 V or 4 mA, resp.)**

![Graph showing leakage characteristics.]

- Flow in l/min
- Pressure differential in bar
- Spread

The area ratio of the orifice spool has an influence on the actuating time as follows:
**Characteristic curves** (measured with HLP 46 and $\theta_{oil} = 40^\circ\text{C} \pm 5^\circ\text{C})$

**Flow characteristic linear**

FES(E) 32 C.../450L... direction of flow A → B

![Graph showing flow characteristic linear for direction A → B]

1. $\Delta p = 5$ bar
2. $\Delta p = 10$ bar
3. $\Delta p = 20$ bar
4. $\Delta p = 30$ bar
5. $\Delta p = 50$ bar
6. $\Delta p = 100$ bar

FES(E) 32 C.../450L... direction of flow B → A

![Graph showing flow characteristic linear for direction B → A]

1. $\Delta p = 5$ bar
2. $\Delta p = 10$ bar
3. $\Delta p = 20$ bar
4. $\Delta p = 30$ bar
5. $\Delta p = 50$ bar
6. $\Delta p = 100$ bar

**Transient function with stepped command value change**

![Graph showing step responses for transient function]

Step responses:
- 0 → 100 → 0% (solid line)
- 10 → 90 → 10% (dashed line)
- 25 → 75 → 25% (dotted line)

1. Measurement conditions:
   - Pressure in A = 50 bar
   - Verbraucher in B geschlossen ($p_A = p_B = 50$ bar)
   - Pressure in A < 50 bar → actuating time is extended
   - Pressure in A > 50 bar → actuating time is shortened

The area ratio of the orifice spool has an influence on the actuating time as follows:
- Command value 0 → 100%: The actuating time becomes shorter, the higher the inlet pressure and the smaller the $\Delta p$ across the valve.
- Command value 100 → 0%: The actuating time becomes shorter, the higher the inlet pressure and the higher the $\Delta p$ across the valve.

**Leakage from A → B and B → A in dependence upon the pressure differential $\Delta p$ (command value 0 V or 4 mA, resp.)**

![Graph showing leakage from A → B and B → A]

- $\Delta p$ in bar
- Flow in l/min

Spread

Pressure differential in bar → Flow in l/min
Characteristic curves (measured with HLP 46 and $\vartheta_{\text{oil}} = 40 ^\circ \text{C} \pm 5 ^\circ \text{C}$)

Flow characteristic linear

FES(E) 40 C…/670L… Direction of flow A → B

<table>
<thead>
<tr>
<th>Δp</th>
<th>Flow in l/min</th>
</tr>
</thead>
<tbody>
<tr>
<td>5 bar</td>
<td>1120</td>
</tr>
<tr>
<td>10 bar</td>
<td>960</td>
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<tr>
<td>20 bar</td>
<td>800</td>
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<tr>
<td>30 bar</td>
<td>640</td>
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<tr>
<td>50 bar</td>
<td>480</td>
</tr>
<tr>
<td>100 bar</td>
<td>320</td>
</tr>
</tbody>
</table>

FES(E) 40 C…/670L… Direction of flow B → A

<table>
<thead>
<tr>
<th>Δp</th>
<th>Flow in l/min</th>
</tr>
</thead>
<tbody>
<tr>
<td>5 bar</td>
<td>1120</td>
</tr>
<tr>
<td>10 bar</td>
<td>960</td>
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<tr>
<td>20 bar</td>
<td>800</td>
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<tr>
<td>30 bar</td>
<td>640</td>
</tr>
<tr>
<td>50 bar</td>
<td>480</td>
</tr>
<tr>
<td>100 bar</td>
<td>320</td>
</tr>
</tbody>
</table>

Transient function with stepped command value change

Step responses

<table>
<thead>
<tr>
<th>Stroke in %</th>
<th>Time in ms</th>
</tr>
</thead>
<tbody>
<tr>
<td>0 – 100 – 0%</td>
<td>0 – 100 – 0%</td>
</tr>
<tr>
<td>10 – 90 – 10%</td>
<td>10 – 90 – 10%</td>
</tr>
<tr>
<td>25 – 75 – 25%</td>
<td>25 – 75 – 25%</td>
</tr>
</tbody>
</table>

1) Measurement conditions
Pressure in A = 50 bar
Verbraucher in B geschlossen ($p_A = p_B = 50$ bar)
Pressure in A < 50 bar → actuating time is extended
Pressure in A > 50 bar → actuating time is shortened

The area ratio of the orifice spool has an influence on the actuating time as follows:

→ Command value 0 → 100%: The actuating time becomes shorter, the higher the inlet pressure and the smaller the $\Delta p$ across the valve.

→ Command value 100 → 0%: The actuating time becomes shorter, the higher the inlet pressure and the higher the $\Delta p$ across the valve.

Leakage from A → B and B → A in dependence upon the pressure differential $\Delta p$ (command value 0 V or 4 mA, resp.)

<table>
<thead>
<tr>
<th>Pressure differential in bar</th>
<th>Flow in l/min</th>
</tr>
</thead>
<tbody>
<tr>
<td>0</td>
<td>1.0</td>
</tr>
<tr>
<td>50</td>
<td>2.0</td>
</tr>
<tr>
<td>150</td>
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<tr>
<td>250</td>
<td>4.0</td>
</tr>
<tr>
<td>350</td>
<td>5.0</td>
</tr>
<tr>
<td>450</td>
<td>6.0</td>
</tr>
</tbody>
</table>
**Characteristic curves** (measured with HLP 46 and \( \theta_{\text{oil}} = 40 \, ^\circ\text{C} \pm 5 \, ^\circ\text{C} \))

**Flow characteristic linear**

FES(E) 50 C.../1400L... direction of flow A \( \rightarrow \) B

1. \( \Delta p = 5 \) bar
2. \( \Delta p = 10 \) bar
3. \( \Delta p = 20 \) bar
4. \( \Delta p = 30 \) bar
5. \( \Delta p = 50 \) bar
6. \( \Delta p = 100 \) bar

**FES(E) 50 C.../1400L... direction of flow B \( \rightarrow \) A

1. \( \Delta p = 5 \) bar
2. \( \Delta p = 10 \) bar
3. \( \Delta p = 20 \) bar
4. \( \Delta p = 30 \) bar
5. \( \Delta p = 50 \) bar
6. \( \Delta p = 100 \) bar

**Transient function with stepped command value change**

Step responses

- 0 – 100 – 0 %
- 10 – 90 – 10 %
- 25 – 75 – 25 %

1) Flow values above 1200 l/min are no measured values!

2) Measurement conditions

- Pressure in A = 50 bar
- Verbraucher in B geschlossen (\( \rho_A = \rho_B = 50 \) bar)
- Pressure in A < 50 bar \( \rightarrow \) actuating time is extended
- Pressure in A > 50 bar \( \rightarrow \) actuating time is shortened

The area ratio of the orifice spool has an influence on the actuating time as follows:

\( \rightarrow \) Command value 0 \( \rightarrow \) 100%: The actuating time becomes shorter, the higher the inlet pressure and the smaller the \( \Delta p \) across the valve.

\( \rightarrow \) Command value 100 \( \rightarrow \) 0%: The actuating time becomes shorter, the higher the inlet pressure and the higher the \( \Delta p \) across the valve.

**Leakage from A \( \rightarrow \) B and B \( \rightarrow \) A in dependence upon pressure differential \( \Delta p \) (command value 0 V or 4 mA, resp.)**

- \( \Delta p = 5 \) bar
- \( \Delta p = 10 \) bar
- \( \Delta p = 20 \) bar
- \( \Delta p = 30 \) bar
- \( \Delta p = 50 \) bar
- \( \Delta p = 100 \) bar
Characteristic curves (measured with HLP 46 and $\theta_{\text{oil}} = 40 \, ^\circ\text{C} \pm 5 \, ^\circ\text{C}$)

Flow characteristic linear 1)

FES(E) 63 C…/800L… direction of flow A → B

Flow in l/min

Command value in % →

1. $\Delta p = 5$ bar
2. $\Delta p = 10$ bar
3. $\Delta p = 20$ bar
4. $\Delta p = 30$ bar
5. $\Delta p = 50$ bar
6. $\Delta p = 100$ bar

FES(E) 63C…/800L… Direction of flow B → A

Flow in l/min

Command value in % →

1. $\Delta p = 5$ bar
2. $\Delta p = 10$ bar
3. $\Delta p = 20$ bar
4. $\Delta p = 30$ bar
5. $\Delta p = 50$ bar
6. $\Delta p = 100$ bar

Transient function with stepped command value change 2)

Stroke in %

Time in ms →

Step responses
0 - 100 - 0 %
10 - 90 - 10 %
25 - 75 - 25 %

1) Flow values above 1200 l/min are no measured values!
2) Measurement conditions
   - Pressure in A = 50 bar
   - Actuator in B closed ($p_A = p_B = 50$ bar)
   - Pressure in A < 50 bar → actuating time is extended
   - Pressure in A > 50 bar → actuating time is shortened
   - The area ratio of the orifice spool has an influence on the actuating time as follows:
     - Command value 0 → 100%: the actuating time becomes shorter, the higher the inlet pressure and the smaller the $\Delta p$ across the valve.
     - Command value 100 → 0%: The actuating time becomes shorter, the higher the inlet pressure and the higher the $\Delta p$ across the valve.
**Characteristic curves** (measured with HLP 46 and $\theta_{oil} = 40 ^\circ C \pm 5 ^\circ C$) **Size 63**

Leakage from A $\rightarrow$ B and B $\rightarrow$ A in dependence upon the pressure differential $\Delta p$ (command value 0 V or 4 mA, resp.)

![Graph showing characteristic curves](image)

**Unit dimensions: Type FES** (nominal dimensions in mm)

<table>
<thead>
<tr>
<th>Size</th>
<th>25</th>
<th>32</th>
<th>40</th>
<th>50</th>
<th>63</th>
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</thead>
<tbody>
<tr>
<td>H11</td>
<td>51</td>
<td>63</td>
<td>62</td>
<td>73</td>
<td>90</td>
</tr>
<tr>
<td>H12</td>
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<td>128</td>
<td>127</td>
<td>138</td>
<td>155</td>
</tr>
<tr>
<td>H13</td>
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<td>137.5</td>
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<td>148.5</td>
<td>159.5</td>
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<td>H16</td>
<td>25</td>
<td>35</td>
<td>45</td>
<td>45</td>
<td>65</td>
</tr>
<tr>
<td>L1</td>
<td>85</td>
<td>102.5</td>
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<td>L2</td>
<td>93.5</td>
<td>102.5</td>
<td>126</td>
<td>140</td>
<td>180</td>
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<tr>
<td>L3</td>
<td>42.5</td>
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<td>63</td>
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<td>90</td>
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<td>L8</td>
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<td>219</td>
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<tr>
<td>L9</td>
<td>15</td>
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<td>15</td>
<td>15</td>
<td>15</td>
</tr>
</tbody>
</table>

Required surface quality of mating part

- $0.01/100\text{mm}$
- Rzmax 4

1. Nameplate
2. Identical seal rings for ports X and Y
3. 4 off valve fixing screws to ISO 4762-10.9 (friction coefficient 0.09 ... 0.14 to VDA 235-101) are included in the scope of supply:
   - Size 25: M12 x 60, tightening torque $M_t = 75 \text{ Nm}$
   - Size 32: M16 x 75, tightening torque $M_t = 170 \text{ Nm}$
   - Size 40: M20 x 80, tightening torque $M_t = 350 \text{ Nm}$
   - Size 50: M20 x 90, tightening torque $M_t = 380 \text{ Nm}$
   - Size 63: M30 x 100, tightening torque $M_t = 1200 \text{ Nm}$
4. Cable socket for proportional solenoid, separate order see, page 6
5. Cable socket for inductive position transducer, separate order, see page 6
6. Space required to remove cable socket
Unit dimensions: Type FESE (nominal dimensions in mm)

<table>
<thead>
<tr>
<th>Size</th>
<th>25</th>
<th>32</th>
<th>40</th>
<th>50</th>
<th>63</th>
</tr>
</thead>
<tbody>
<tr>
<td>H11</td>
<td>51</td>
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<td>62</td>
<td>73</td>
<td>90</td>
</tr>
<tr>
<td>H12</td>
<td>116</td>
<td>128</td>
<td>127</td>
<td>138</td>
<td>155</td>
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<tr>
<td>H13</td>
<td>110</td>
<td>122</td>
<td>121</td>
<td>132</td>
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</tr>
<tr>
<td>H16</td>
<td>279</td>
<td>291</td>
<td>290</td>
<td>301</td>
<td>318</td>
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<td>H17</td>
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<td>45</td>
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<tr>
<td>L1</td>
<td>85</td>
<td>102.5</td>
<td>126</td>
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<td>180</td>
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<tr>
<td>L2</td>
<td>93.5</td>
<td>102.5</td>
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<td>180</td>
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<tr>
<td>L3</td>
<td>42.5</td>
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<td>L9</td>
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<td>18.75</td>
<td>30.5</td>
<td>37.5</td>
<td>57.5</td>
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</tbody>
</table>

Required surface quality of mating part

1 Nameplate
2 Identical seal rings for ports X and Y
3 4 off valve fixing screws to ISO 4762-10.9 (friction coefficient 0.09 ... 0.14 to VDA 235-101) are included in the scope of supply:
   Size 25: M12 x 60, tightening torque $M_t = 75 \text{ Nm}$
   Size 32: M16 x 75, tightening torque $M_t = 170 \text{ Nm}$
   Size 40: M20 x 80, tightening torque $M_t = 350 \text{ Nm}$
   Size 50: M20 x 90, tightening torque $M_t = 380 \text{ Nm}$
   Size 63: M30 x 100, tightening torque $M_t = 1200 \text{ Nm}$
4 Cable socket separate order, see page 7
5 Space required to remove cable socket
### Installation dimensions (nominal dimensions in mm)

<table>
<thead>
<tr>
<th>Size</th>
<th>ØD1(^{\text{H8}})</th>
<th>ØD2</th>
<th>ØD3</th>
<th>max. ØD3</th>
<th>ØD4(^{\text{H8}})</th>
<th>D5</th>
<th>max. ØD6</th>
<th>ØD7(^{\text{H13}})</th>
<th>H1</th>
<th>H1 (^{1})</th>
<th>H2</th>
<th>H3</th>
<th>H4</th>
<th>H5</th>
<th>H6</th>
<th>H7</th>
<th>H8</th>
<th>min. H9, (ref. dimension)</th>
<th>min. H10</th>
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<td></td>
<td></td>
<td></td>
<td>1</td>
<td>8</td>
</tr>
</tbody>
</table>

\(^{1}\) Bore centre at max. ØD3

### Tolerances to: General tolerances ISO 2768-mK

7 Port X
8 Port Y
9 Locating bore for locating pin
10 Depth of fit
11 Reference dimension
12 Port B can optionally arranged around the central axis of port A. However, care must be taken not to drill the fixing bores and the pilot bores.
13 In the case of a diameter for port B other than specified in the dimensional table, the distance from the cover contact face to the centre of the bore must be calculated.